



A New Concept to Designing a Combined Cycle Cogeneration Power Plant

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PLAN

1. Introduction
2. Conventional Steam Cogeneration Plant
3. Synthesis of the S-CO₂ Technology with CHP Principles
4. Study methodology

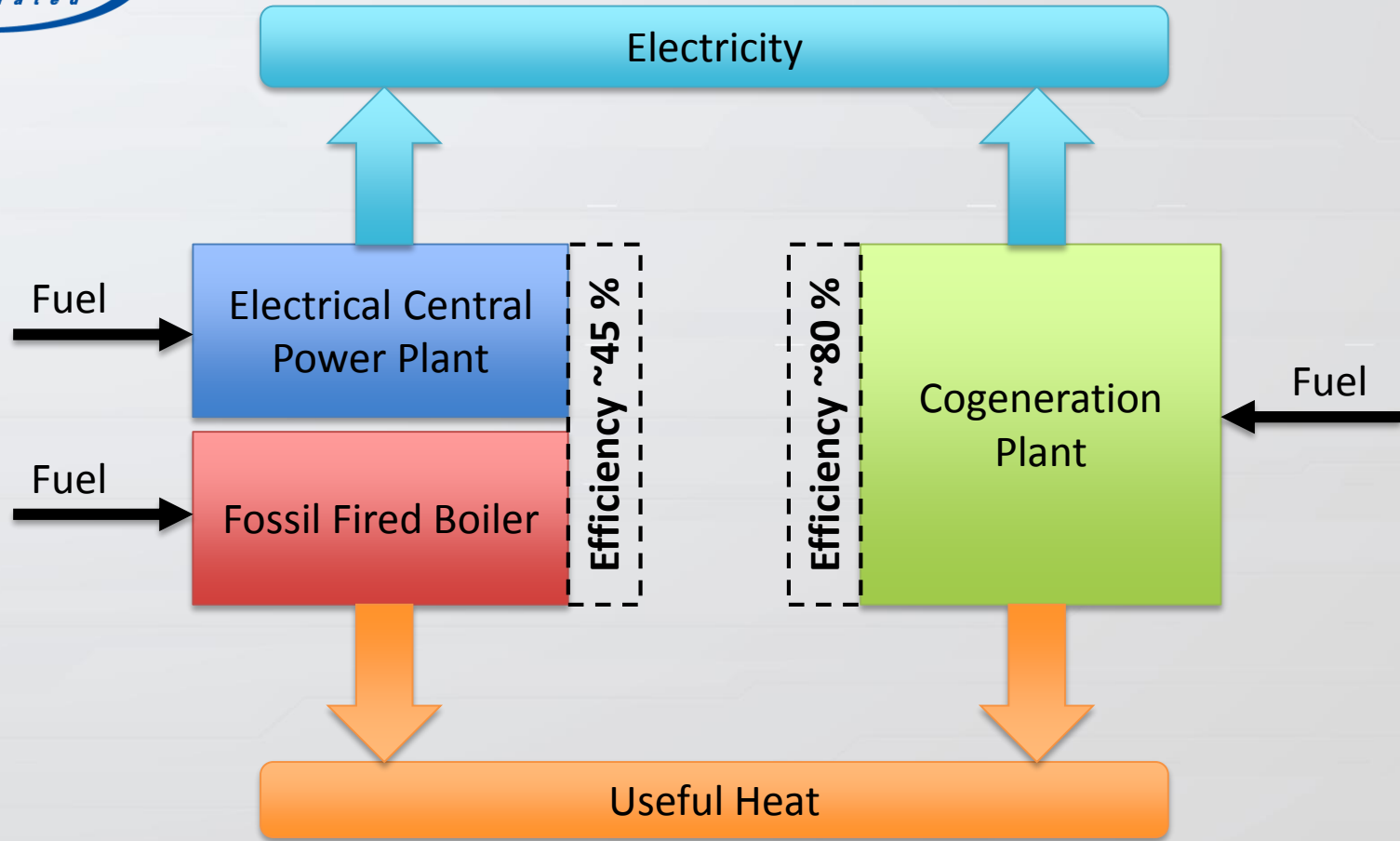


PLAN

5. Combined Complex Steam-S-CO₂ CHP Plant Analysis
6. Cascaded Supercritical CO₂ CHP Plant Analysis
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8. Conclusions



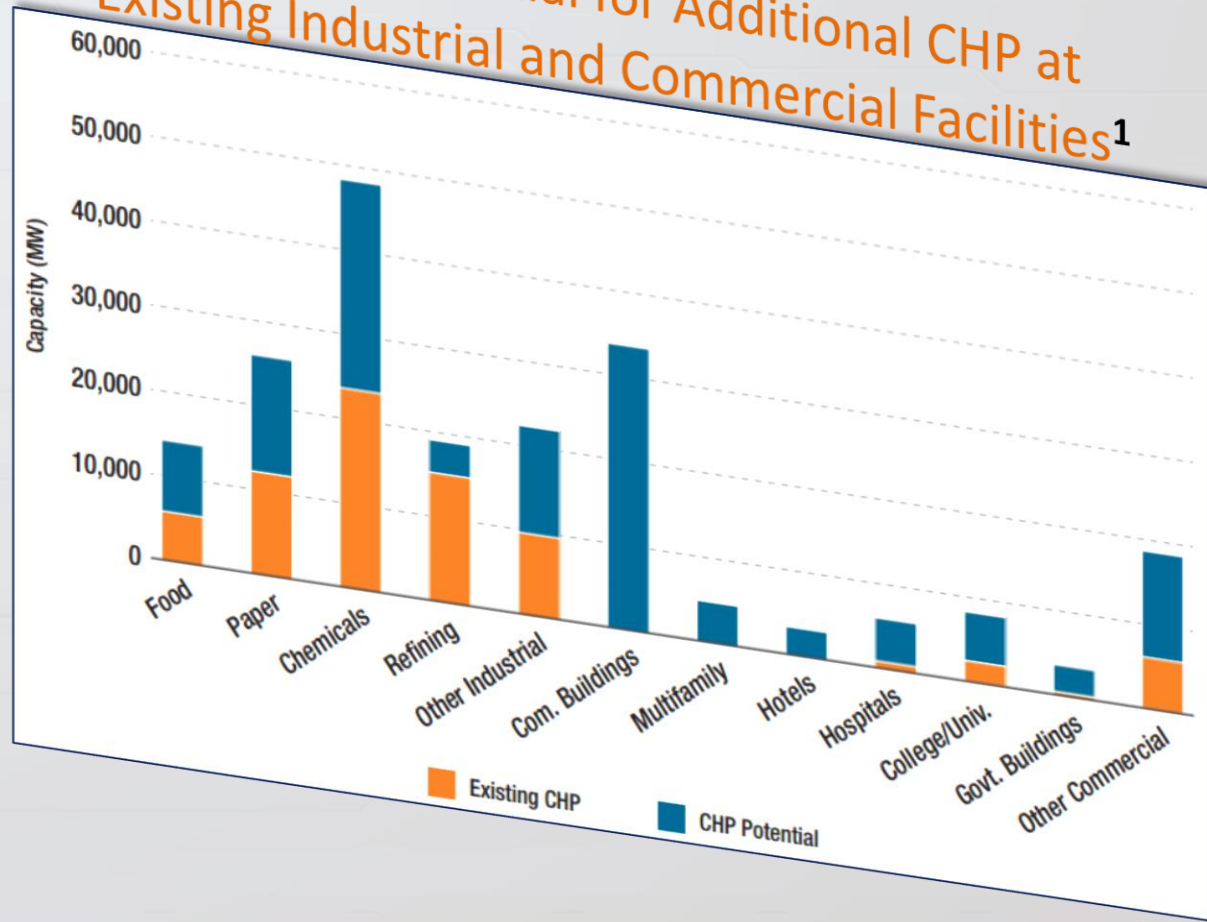
1. Introduction



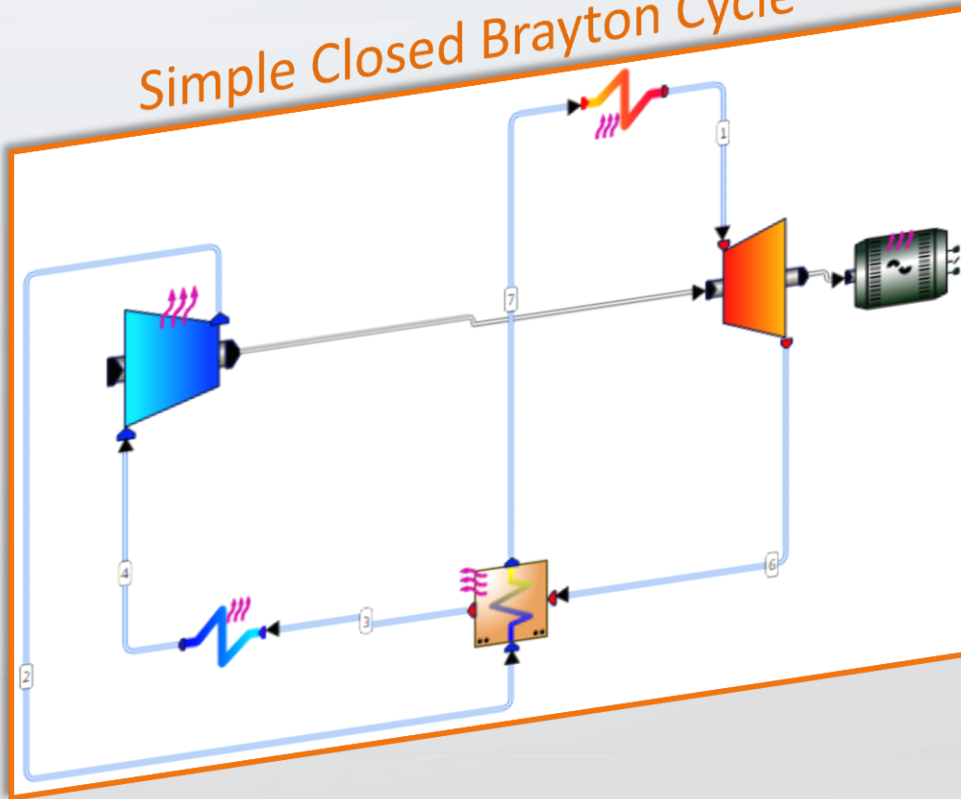


Internal estimates by ICF International and CHP Installation Database developed by ICF International for Oak Ridge National Laboratory and the DOE; 2012

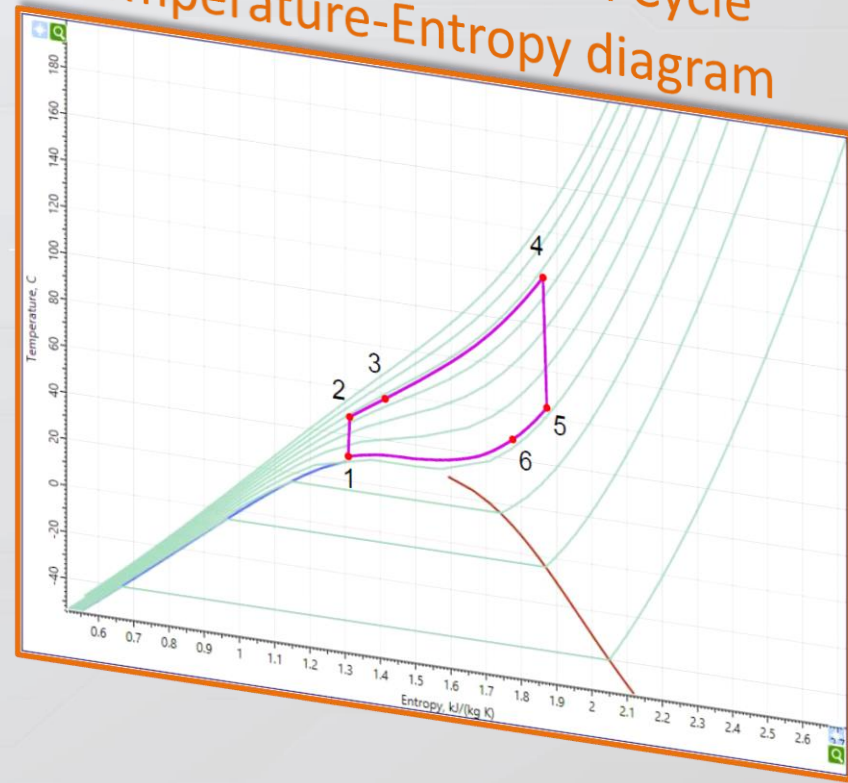
Technical Potential for Additional CHP at Existing Industrial and Commercial Facilities¹



Simple Closed Brayton Cycle

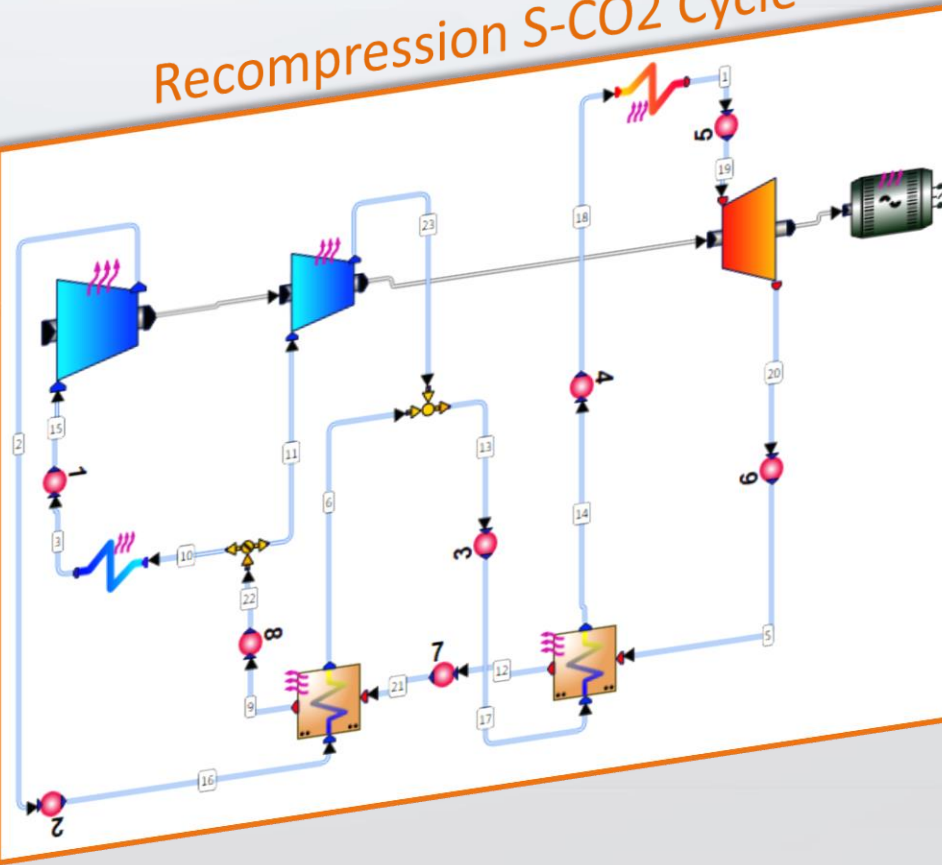


Simple Closed Brayton Cycle Temperature-Entropy diagram

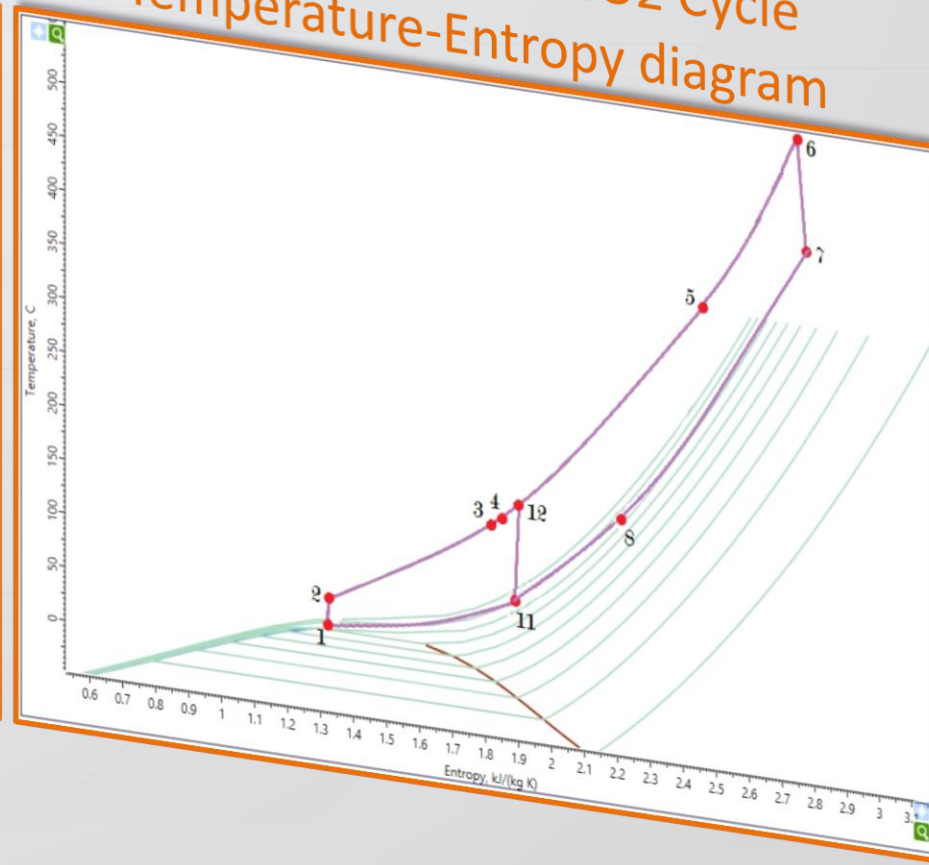




Recompression S-CO₂ Cycle

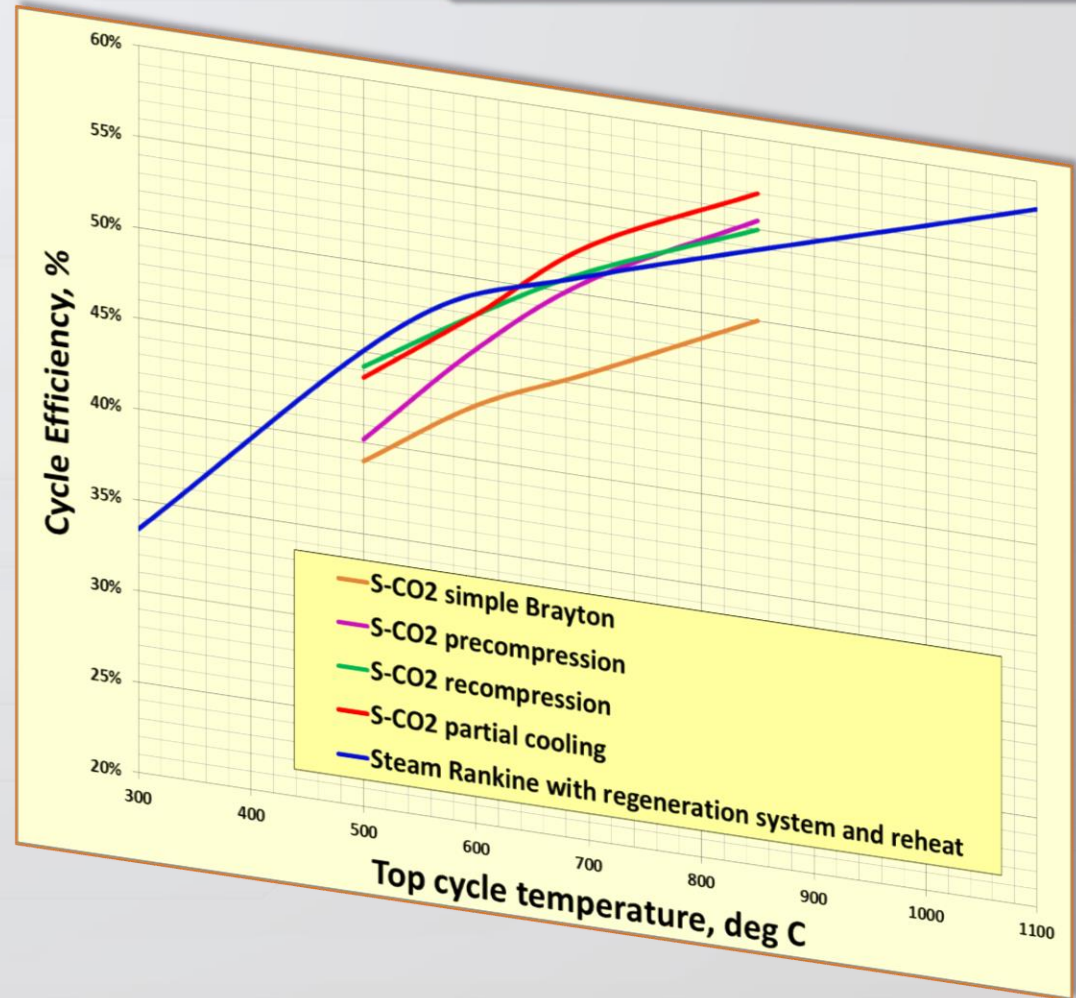


Recompression S-CO₂ Cycle Temperature-Entropy diagram





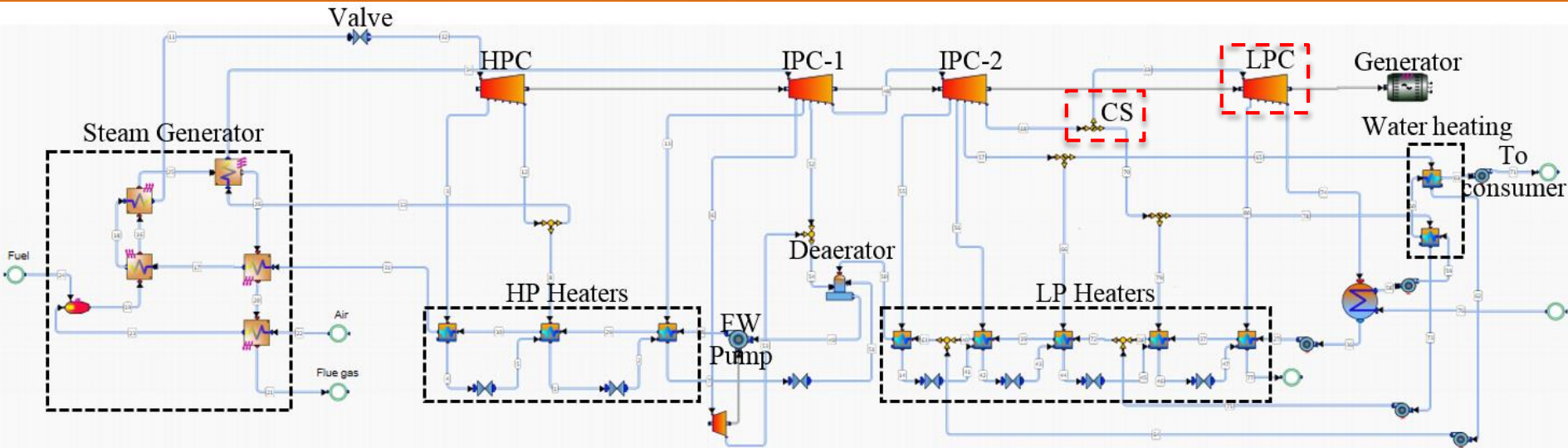
Cycle Efficiency
Dependencies of
different S-CO₂ cycles
and Steam Rankine cycle
from live fluid
temperature





2. Conventional Steam Cogeneration Plant

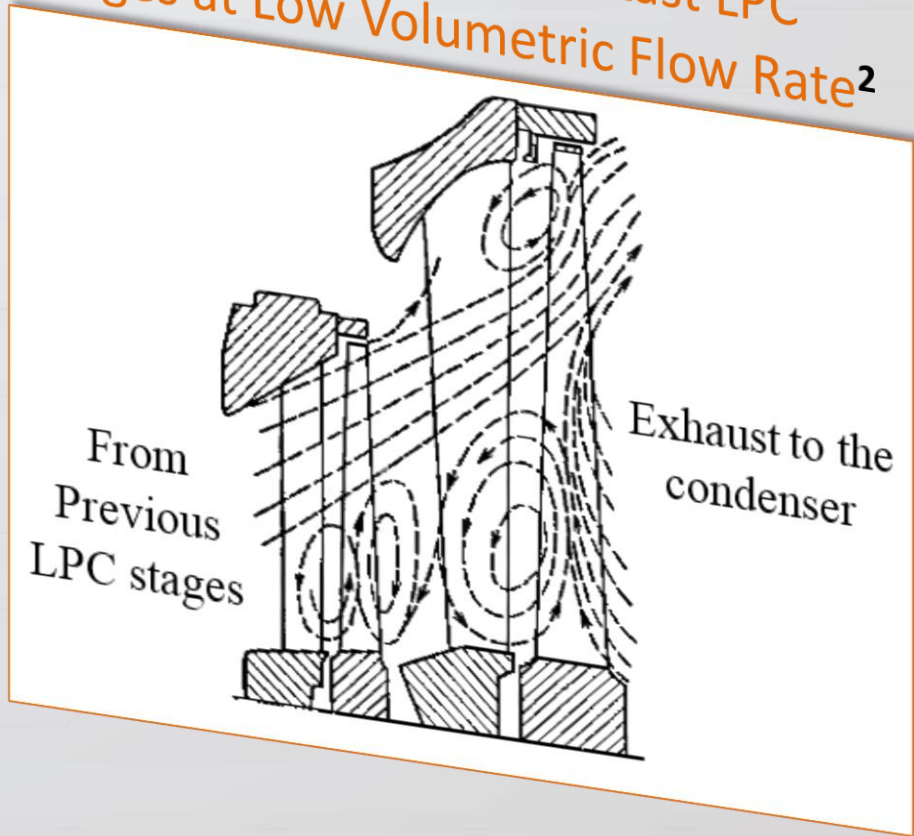
Schematic Flow Diagram of T-250/300-23.5 CHP unit





- Flow separations near hub and tip at low mass flow rate
- Part of the main flow energy is lost
- Captured energy dissipates into the heat, which, in turn, increases the temperature of the LPC flow path

Flow Structure in the Last LPC Stages at Low Volumetric Flow Rate²





3. Synthesis of the S-CO₂ Technology with CHP Principles



1

**Steam Rankine cycle
CHP plant with
bottoming supercritical
CO2 cycle**

Embodiment a.
Combined
Complex
Steam-S-CO2
CHP Plant

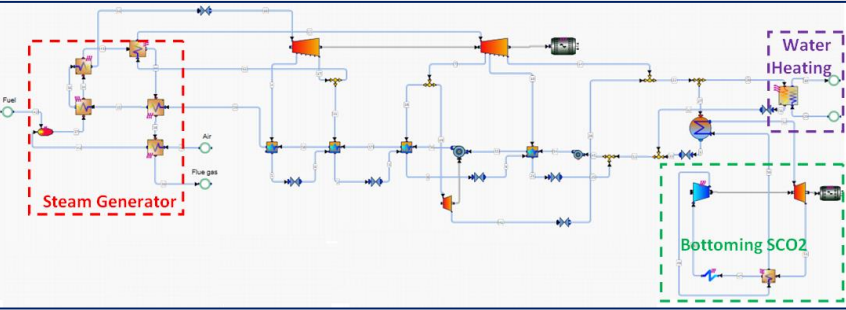
Embodiment b.
Combined
Simple Steam-
S-CO2 CHP
Plant

2

**CHP plant with single
supercritical CO2
working fluid**

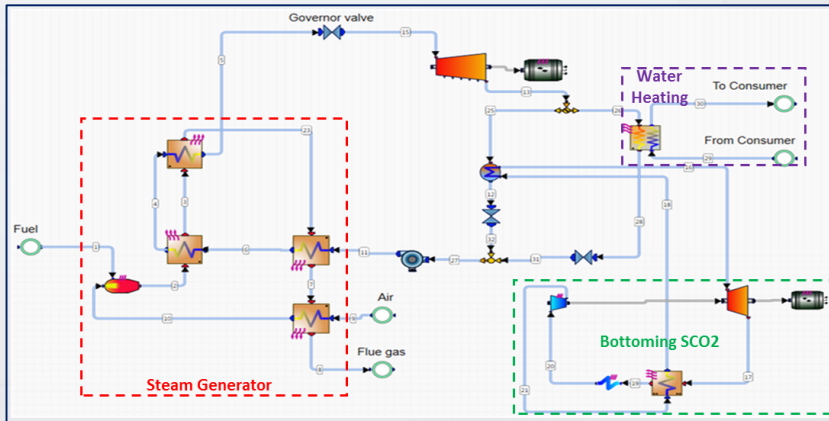
Embodiment a.
Cascaded
Supercritical
CO2 CHP Plant

Embodiment b.
Single
Supercritical
CO2 CHP Plant



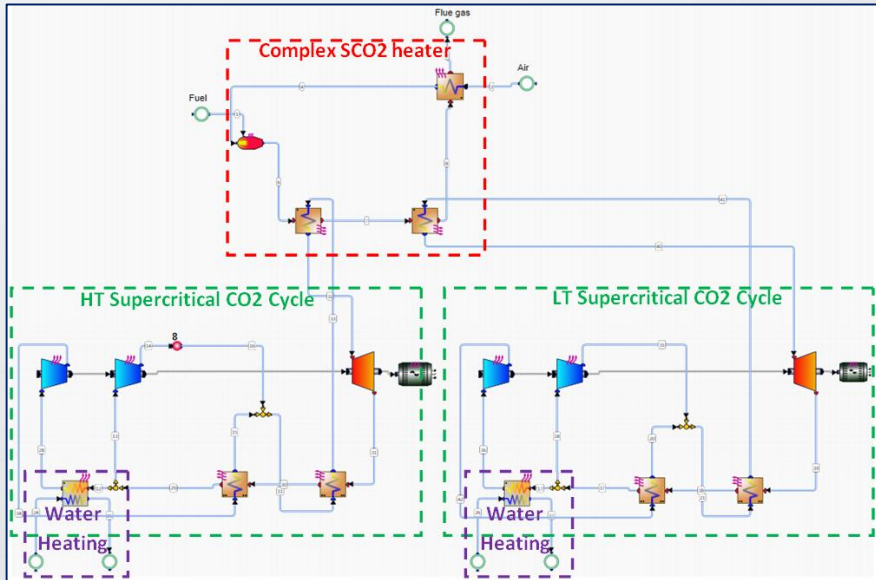
Embodiment 1a Combined Complex Steam-SCO2 CHP Plant

	Parameter	Value
Steam part		
1	Live steam pressure, MPa	23.5
2	Live steam temperature, °C	540
3	Back pressure, MPa	0.2
4	Constant net electrical power production, MW	252.3
Bottoming SCO2 part		
5	Compressor outlet pressure, MPa	15
6	Maximum supercritical CO2 temperature, °C	120
7	CO2 cycle lower pressure, MPa	7.7
8	CO2 temperature after the cooler, °C	32



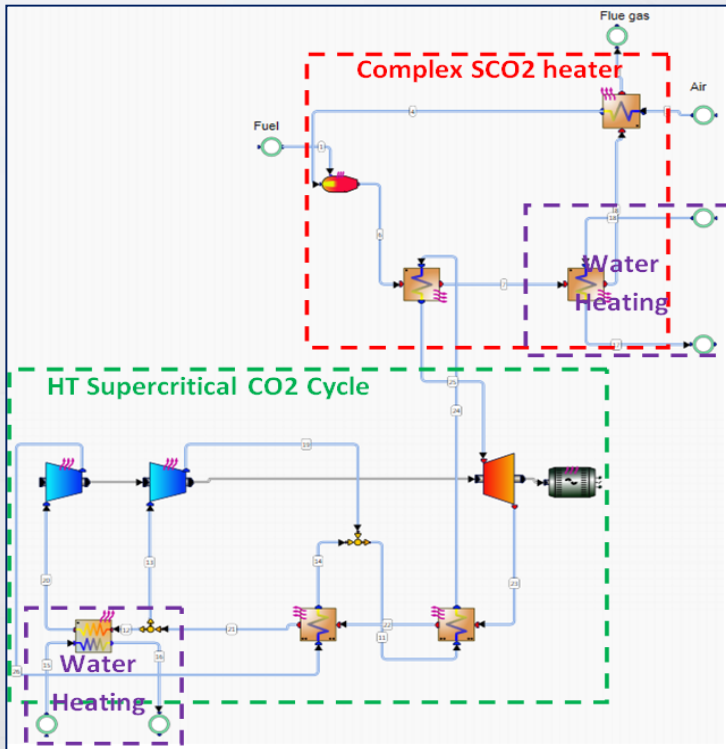
Embodiment 1b Combined Simple Steam-SCO2 CHP Plant

	Parameter	Value
Steam part		
1	Live steam pressure, MPa	23.5
2	Live steam temperature, °C	540
3	Pressure at steam turbine outlet, MPa	3
4	Constant net electrical power production, MW	140.4
Bottoming SCO2 part		
5	Compressor outlet pressure, MPa	25
6	Maximum supercritical CO2 temperature, °C	225
7	CO2 cycle lower pressure, MPa	7.7
8	CO2 temperature after the cooler, °C	32



Embodiment 2a Cascaded SCO2 CHP Plant

	Parameter	HT cycle	LT cycle
1	Live CO2 temperature, °C	540	300
2	Maximum cycle pressure, MPa	21	20
3	Lower cycle pressure, MPa	7.7	7.7
4	CO2 temperature at LT recuperator outlet, °C	115	110
5	Temperature at CO2 cooler (water heater) outlet, °C	37	37
6	Net electrical power production, MW	297.7	13.9
7	Hot water temperature, °C	100	



Embodiment 2b

Single SCO2 CHP Plant

	Parameter	HT cycle
1	Live CO2 temperature, °C	540
2	Maximum cycle pressure, MPa	21
3	Lower cycle pressure, MPa	7.7
4	CO2 temperature at LT recuperator outlet, °C	115
5	Temperature at CO2 cooler (water heater) outlet, °C	37
6	Net electrical power production, MW	297.7
7	Hot water temperature, °C	100



Performances of the considered CHP plants

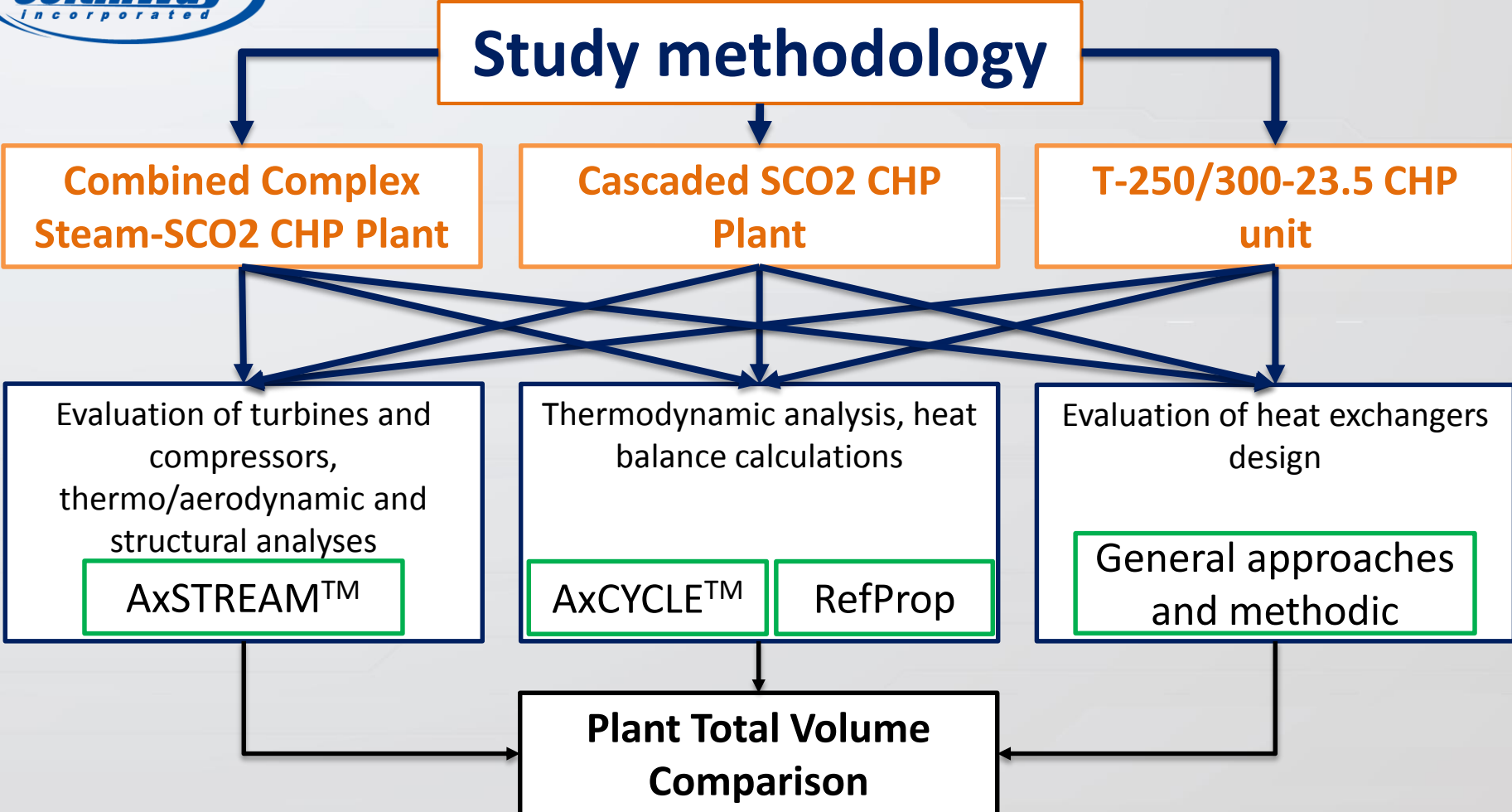
Parameter	Embodiment 1.a		Embodiment 1.b		Embodiment 2.a		Embodiment 2.b		T-250/300	
	0	100	0	100	0	100	0	100	0	100
Heat load, %	0	100	0	100	0	100	0	100	0	100
Heat consumption, MW	789.998									
Net electrical power, MW	304.4	252.2	266.6	140.4	311.6	311.6	297.7	297.7	300	250
Useful heat, MW	-	494.6	-	607.6	-	421.0	-	436.2	-	384.1
Electrical efficiency, %	38.53	31.93	33.76	17.77	39.44	39.4	37.68	37.68	37.97	31.65
Heat utilization factor	0.385	0.945	0.338	0.947	0.394	0.927	0.364	0.929	0.380	0.803
Potential of Useful heat, MW	-	-	-	-	421.0	-	436.2	-	-	-



- S-CO₂ CHP plants have an advantage at moderate live fluid temperature – 540 °C
- The synthesis of the S-CO₂ and CHP technologies gives more benefit than pure electricity production S-CO₂ embodiments



4. Study methodology

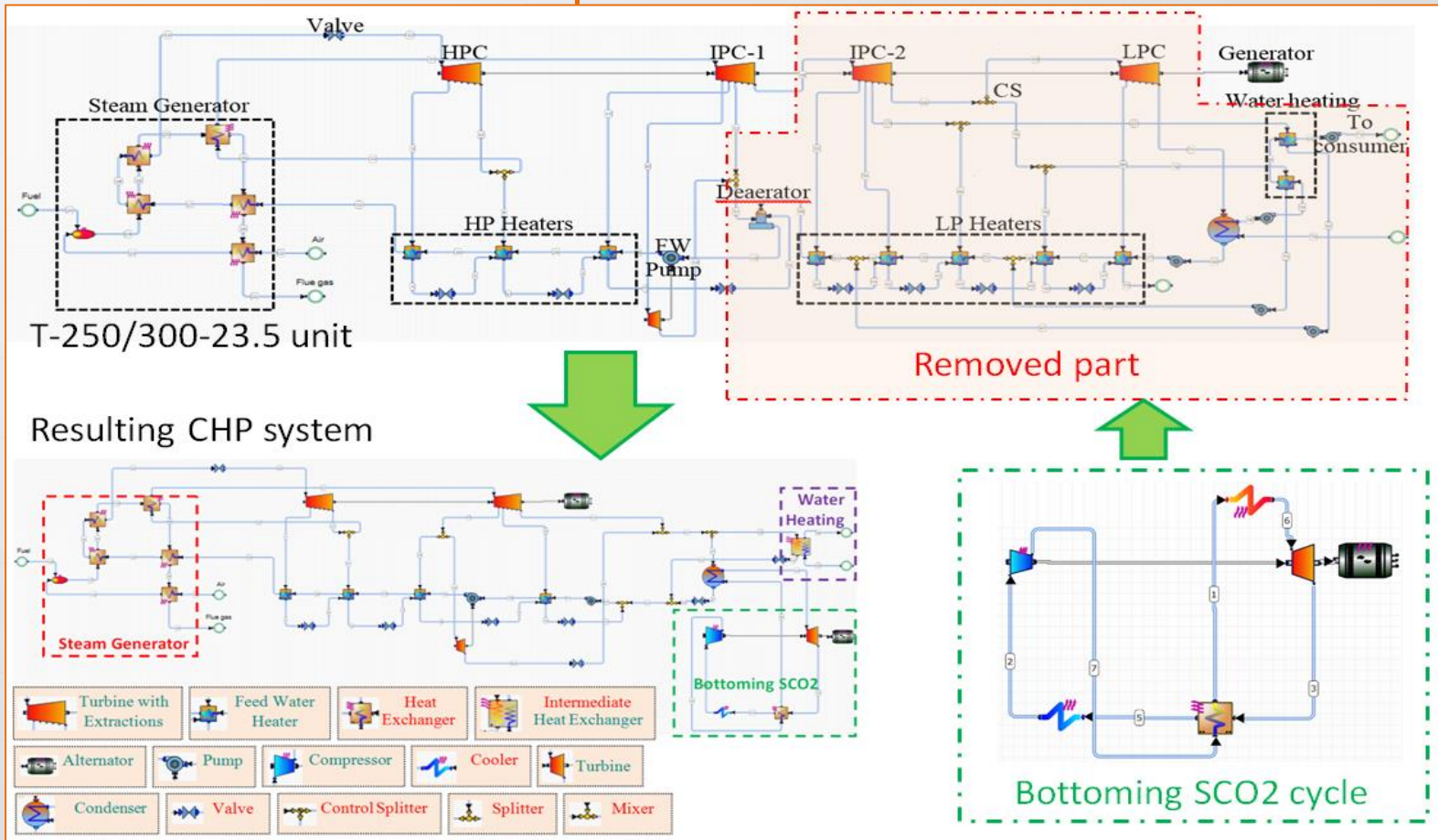




5. Combined Complex Steam-S-CO₂ CHP Plant Analysis

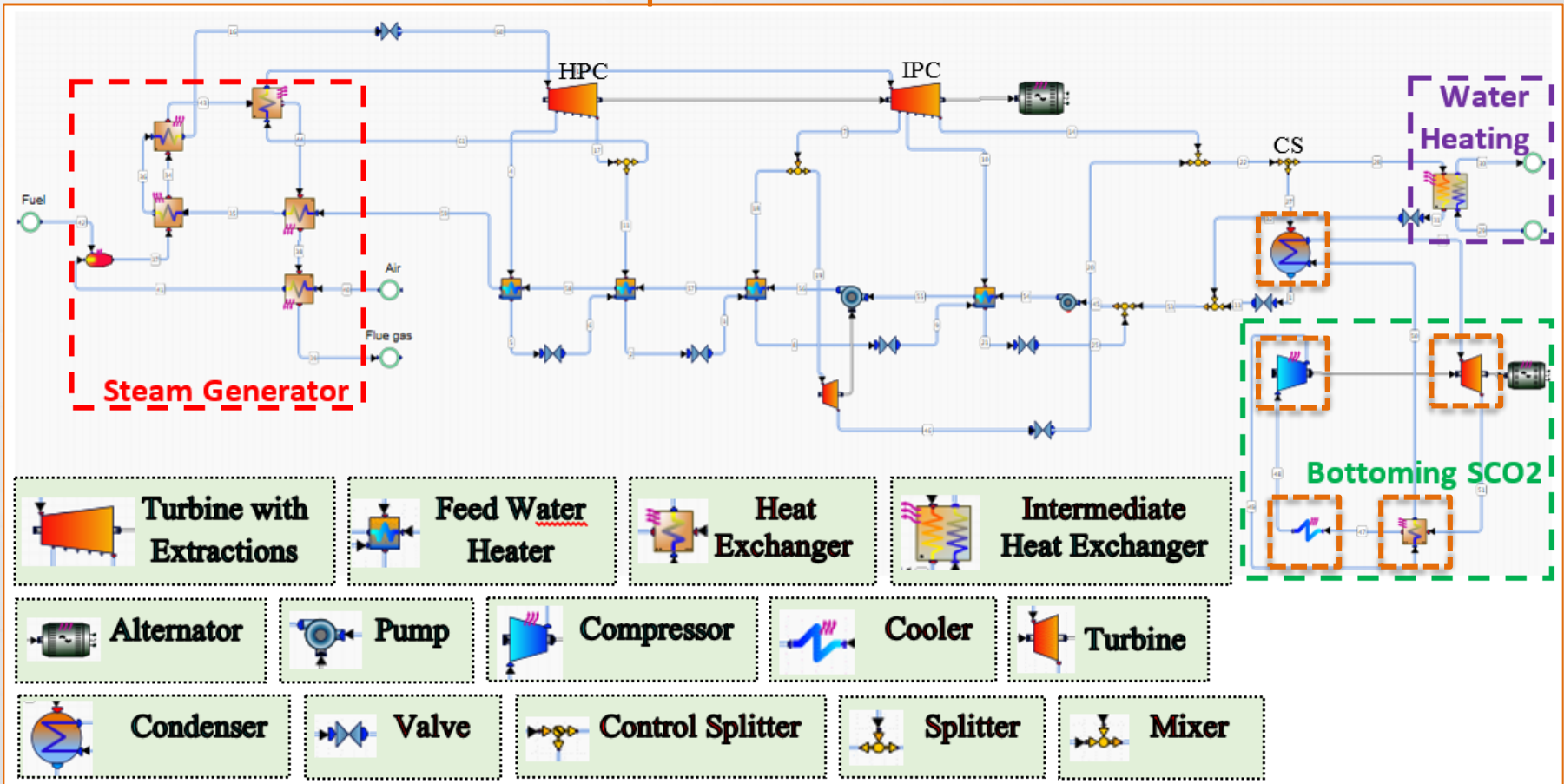


Combined Complex Steam-SCO₂ CHP Plant





Combined Complex Steam-SCO₂ CHP Plant





Combined Complex Steam-SCO₂ CHP Plant Dimensions: Bottoming S-CO₂ cycle components

Parameter	Turbine	Compressor	Recuperator	Cooler	Condenser
Type	Reaction	Centrifugal	Shell&tube	Shell&tube	Shell&tube
Estimated dimensions, m	D1.4*L1.4	D2.4*L0.5	D3.3*L12.8	D2.7*L11.9	D2.5*L9.3
Volume, m³	2.2	2.3	109.5	68.1	45.7
Total volume, m³	227.7				



6. Cascaded Supercritical CO₂ CHP Plant Analysis



HT Cycle of the Cascaded Supercritical CO2 CHP Plant Dimensions

Parameter	Turbine	Main Compressor	Recompressor	High temperature recuperator	Low temperature recuperator	Water heater
Type	Reaction type	Centrifugal	Centrifugal	Shell&tube	Shell&tube	Shell&tube
Estimated dimensions, m	D1.6*L4.0	D2.5*L0.5	D5.0*L0.8	D3.5*L13.2	D3.3*L11.9	D2.2 *L10.9
Volume, m3	8.0	2.5	15.7	127.0	101.8	41.4
Total volume, m3	296.4					



LT Cycle of the Cascaded Supercritical CO₂ CHP Plant Dimensions

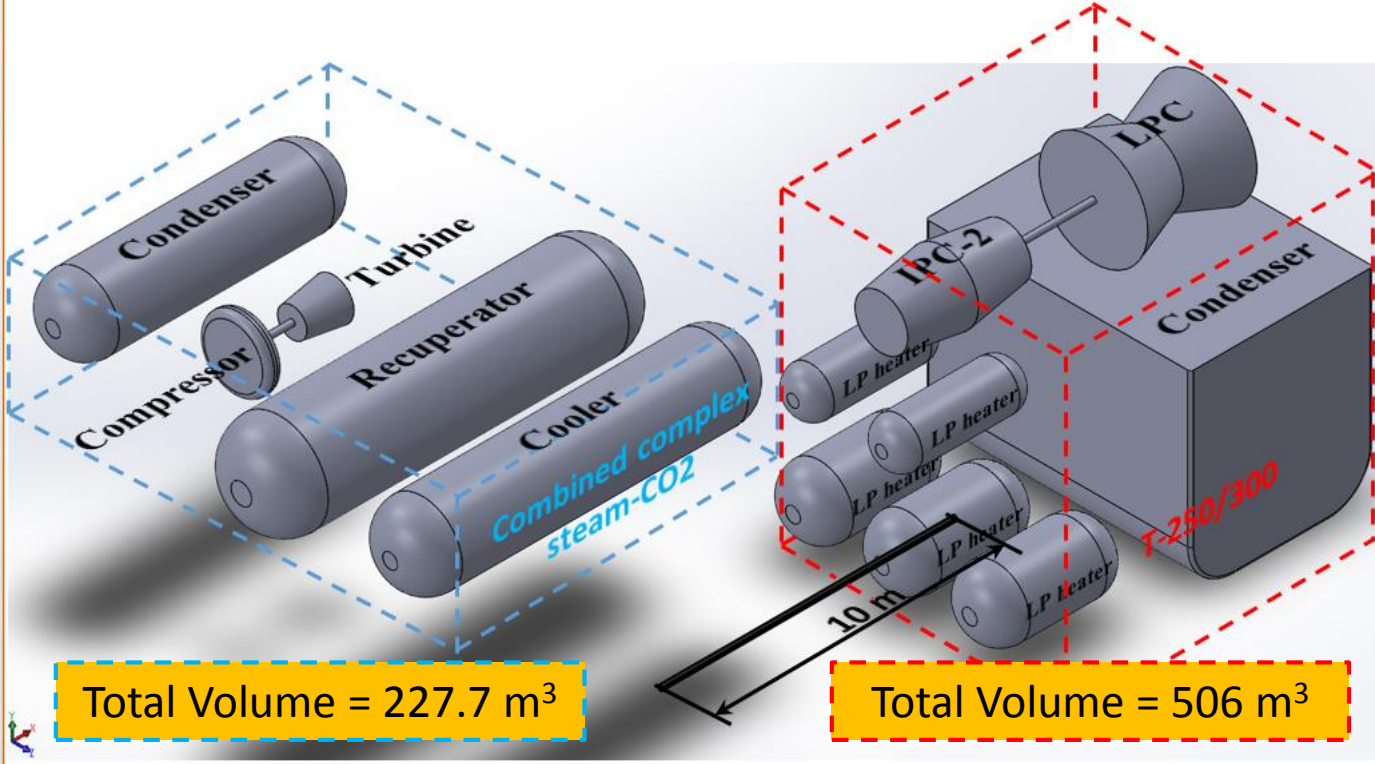
Parameter	Turbine	Main Compressor	Recompressor	High temperature recuperator	Low temperature recuperator	Water heater
Type	Impulse type	Axial	Axial	Shell&tube	Shell&tube	Shell&tube
Estimated dimensions, m	D0.7*L1.6	D0.5*L0.5	D0.8*L0.8	D1.3*L8.5	D1.9*L9.1	D1.3 *L9.3
Volume, m3	0.6	0.1	0.3	11.3	25.8	12.2
Total volume, m3	50.5					



7. Total Dimensions Difference Comparison

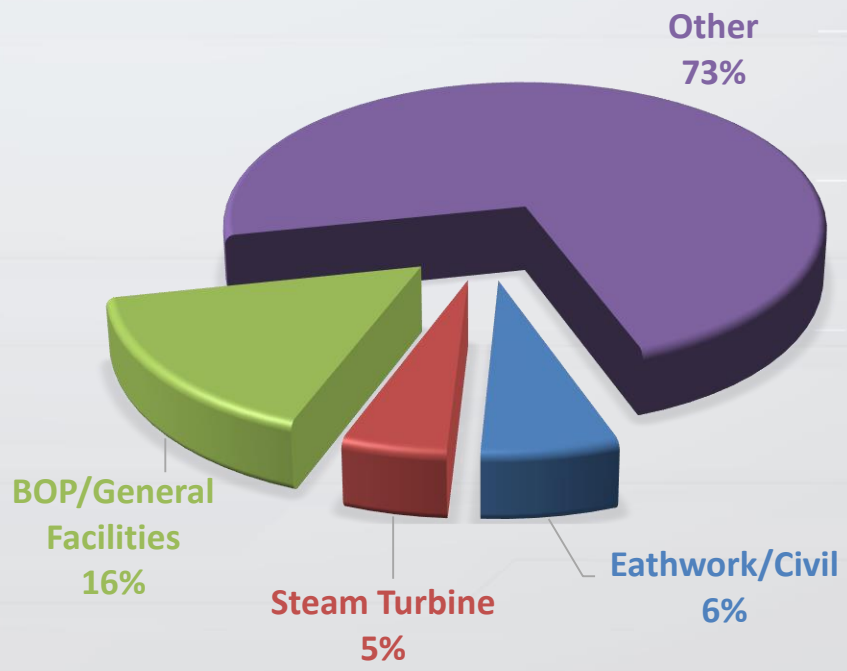


Comparison of the interchangeable components dimensions between the Combined Complex Steam-SCO₂ and the T-250/300





Costs for 300 MW Pulverized Coal Power Plant³



Conceptual Cost Estimate Summary	Coal →	U.S. PRB (thousands \$)
Earthwork/Civil		52,600
Structural Steel		29,400
Mechanical Equipment		
Boiler		113,400
Steam Turbine		40,200
Coal Handling		38,200
Ash Handling		13,400
Particulate Removal System		17,800
Wet Flue Gas Desulfurization (FGD) System		61,800
Selective Catalytic Reduction		26,400
Total Mechanical Equipment		311,200
Electrical		47,200
Piping		32,000
BOP/General Facilities		130,400
Direct Field Cost		602,800
Indirect Costs ¹		46,000
Engineering and Home Office Costs ²		62,600
Process Contingency		0
Project Contingency		106,700
Total Plant Cost		818,100



8. Conclusions



Conclusions

1. Taking into account the benefits of CHP plants and the high potential of the Supercritical CO₂ technology, the latter should also be considered as the basis for future CHP plants. The comparison with traditional Steam based CHP plants should be performed.



Conclusions

2. The most significant drawback of the convenient steam CHP plants scheme is that at a high heat load, the LPC operates at low mass flow rates. At partial mass flow rate modes the LPC may consume the power from the shaft which may lead to an unwanted flow path temperature increase and all the consequent problems.



Conclusions

3. Two new embodiments of the S-CO₂ CHP plant (Combined Complex Steam-S-CO₂ CHP Plant and Cascaded Supercritical CO₂ CHP Plant) are considered and compared with the conventional steam CHP plant in terms of components total sizes.



Conclusions

4. Comparisons of the total volumes of the Combined Complex Steam-SCO₂ and the T-250/300 of the interchangeable components showed that new CO₂ components have in 2.2 times less total volume (506 m³/ 227.7 m³).



Conclusions

5. Cascaded S-CO₂ CHP unit has a total volume 347 m³. This is 2.3 times less than T-250/300-23.5 unit total volume.



Conclusions

6. Taking into account the high effectiveness of Cascaded S-CO₂ CHP plant (311.6 MW of net electrical power and 421 MW of useful heat) at any heat load mode and relatively small total volume, it must be considered a promising CHP plant concept.



Conclusions

7. The embodiment with the bottoming S-CO₂ cycle also may be reasonable in terms of effectiveness and footprint but partial modes of operation remain to be studied.



Sources of the borrowed pictures

1. Combined Heat and Power. A Clean Energy Solution // U.S. Department of Energy, U. S. Environmental Protection Agency, August 2012.
2. Truhnij A., Lomakin B., Cogeneration steam turbines and units: manual for University, MPEI press, 2002. ISBN 5-7046-0722-5
3. Dirk Pauschert, Study of Equipment Prices in the Power Sector, ESMAP Technical Paper 122/09, December 2009.



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